

Scientific Computing I

Module 5: Heat Transfer – Discrete and Continuous Models

Tobias Neckel
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Motivation: Heat Transfer

Wiremesh Model

A Finite Volume Model

Time Dependent Model

Motivation: Heat Transfer

- objective: compute the temperature distribution of some object
- under certain prerequisites:
 - temperature at object boundaries given
 - heat sources
 - material parameters
- observation from physical experiments:

$$q \approx k \cdot \delta T$$

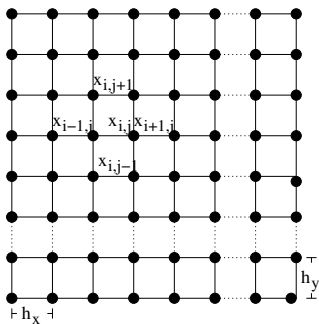
heat flux proportional to temperature differences

Part I

Discrete Models

A Wiremesh Model

- consider rectangular plate as fine mesh of wires
- compute temperature $x_{i,j}$ at nodes of the mesh



A Wiremesh Model (3)

- temperature known at (part of) the boundary; for example

$$x_{0,j} = T_j$$

models a heated/cooled wall with constant temperature T_j at the left boundary.

- temperature flux known at (part of) the boundary; for example

$$x_{i,0} = x_{i,1} \Leftrightarrow x_{i,1} - x_{i,0} = 0$$

models an isolated wall at the lower boundary.

- heat sources: temperature given at a certain position i, j :

$$x_{i,j} = T_s.$$

- task: solve Linear System of Equations

A Wiremesh Model (2)

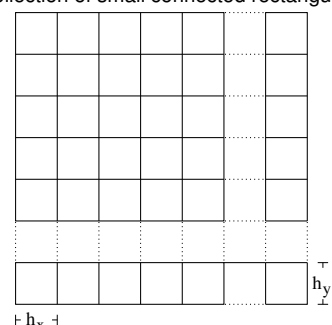
- model assumption: temperatures in equilibrium at every mesh node
- equilibrium: steady state (of temperature), energy balance (inflow = outflow) in each node of the mesh
- incoming temperature fluxes at point i, j via the four wires:
 - from the left: $k (x_{i-1,j} - x_{i,j})$
 - from the right: $k (x_{i+1,j} - x_{i,j})$
 - from below: $k (x_{i,j-1} - x_{i,j})$
 - from above: $k (x_{i,j+1} - x_{i,j})$
- equation for steady state: sum over all fluxes = zero:

$$x_{i,j} = \frac{1}{4} (x_{i-1,j} + x_{i+1,j} + x_{i,j-1} + x_{i,j+1})$$

for all temperatures $x_{i,j}$.

A Finite Volume Model

- object: a rectangular metal plate (again)
- model as a collection of small connected rectangular cells



- examine the heat flux across the cell edges

A Finite Volume Model (2)

- model assumption: temperatures in equilibrium in every grid cell
- heat flux across a given edge is proportional to
 - temperature difference ($T_1 - T_0$) between the adjacent cells
 - length h of the edge
- e.g.: heat flux across the left edge:

$$q_{i,j}^{(\text{left})} = k_x (T_{i,j} - T_{i-1,j}) h_y$$

note: heat flux **out** of the cell (and $k_x > 0$)

- heat flux across all edges determines change of heat energy:

$$q_{ij} = k_x (T_{ij} - T_{i-1,j}) h_y + k_x (T_{ij} - T_{i+1,j}) h_y \\ + k_y (T_{ij} - T_{i,j-1}) h_x + k_y (T_{ij} - T_{i,j+1}) h_x$$

Towards a Time Dependent Model

- idea: set up an ODE for each cell
- simplification: no external heat sources or sinks, i.e. $f_{i,j} = 0$
- change of temperature per time is proportional to heat flux $q_{i,j}(t)$ into the cell (no longer 0):

$$\frac{d}{dt} T_{i,j}(t) = -c \cdot q_{i,j}(t) \\ = \frac{k_x}{h_x} (-2T_{ij}(t) + T_{i-1,j}(t) + T_{i+1,j}(t)) \\ + \frac{k_y}{h_y} (-2T_{ij}(t) + T_{i,j-1}(t) + T_{i,j+1}(t))$$

- solve a system of ODEs

Part II: A Continuous Model – The Heat Equation

From Discrete to Continuous

Derivation of the Heat Equation

Variants of the Heat Equation

Boundary and Initial Conditions

Part II

A Continuous Model – The Heat Equation

From Discrete to Continuous

- remember the discrete model:

$$f_{i,j} = -\frac{k_x}{h_x} (2T_{i,j} - T_{i-1,j} - T_{i+1,j}) \\ -\frac{k_y}{h_y} (2T_{i,j} - T_{i,j-1} - T_{i,j+1})$$

- assumption: heat flux across edges is proportional to temperature *difference*

$$q_{i,j}^{(\text{left})} = k_x (T_{i,j} - T_{i-1,j}) h_y$$

- in reality: heat flux proportional to temperature *gradient*

$$q_{i,j}^{(\text{left})} \approx kh_y \frac{T_{i,j} - T_{i-1,j}}{h_x}$$

A Steady-State Model

- heat sources: consider additional source term $F_{i,j}$ due to
 - external heating
 - radiation
- $F_{i,j} = f_{i,j} h_x h_y$ ($f_{i,j}$ heat flux per area)
- equilibrium with source term requires $q_{i,j} + F_{i,j} = 0$:

$$f_{i,j} h_x h_y = -k_x h_y (2T_{i,j} - T_{i-1,j} - T_{i+1,j}) \\ -k_y h_x (2T_{i,j} - T_{i,j-1} - T_{i,j+1})$$

- again, Linear System of Equations

Boundary Conditions

(Finite Volume Models)

- temperature known in boundary layer cells; for example

$$q_{1,j}^{(\text{left})} = k_x (T_{1,j} - T_{0,j}) h_y = k_x (T_{1,j} - T(x_{0,j})) h_y$$

with $T_{0,j} = T(x_{0,j})$ not an unknown!

(models a heated/cooled wall with constant temperature $T(x_{0,j})$ at the left boundary)

- temperature flux known in boundary layer cells; e.g. $q_{1,j}^{(\text{left})} = 0$:

$$f_{1,j} h_x h_y = -k_x h_y (T_{1,j} - T_{2,j}) \\ -k_y h_x (2T_{1,j} - T_{1,j-1} - T_{1,j+1})$$

models an isolated wall at the left boundary.

Derivation of the Heat Equation

- finite volume model, but with arbitrary control volume D
- change of heat energy (per time) is a result of
 - transfer of heat energy across D 's surface,
 - heat sources and sinks in D (external influences)
- resulting integral equation:

$$\frac{\partial}{\partial t} \int_D \rho c T dV = \int_D q dV + \int_{\partial D} k \nabla T \cdot \vec{n} dS$$

density ρ , specific heat c , and heat conductivity k are material parameters

- heat sources and sinks are modelled in term q

Variants of the Heat Equation

Different scenarios:

- vanishing external influence, $q = 0$:

$$T_t = \kappa \Delta T$$

alternative notation

$$\frac{\partial T}{\partial t} = \kappa \cdot \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right)$$

- equilibrium solution, $T_t = 0$:

$$0 = \kappa \Delta T + \frac{q}{\rho c} \quad \rightarrow \quad -\Delta T = \frac{q}{\kappa \rho c}$$

“Poisson’s Equation”

Part III

Discretization: Finite Difference and Finite Volume Methods

Derivation of the Heat Equation (2)

- according to theorem of Gauß:

$$\int_{\partial D} k \nabla T \cdot \vec{n} dS = \int_D k \Delta T dV$$

- leads to integral equation for **any** domain D :

$$\int_D \rho c T_t - q - k \Delta T dV = 0$$

- hence, the integrand has to be identically 0:

$$T_t = \kappa \Delta T + \frac{q}{\rho c}, \quad \kappa := \frac{k}{\rho c}$$

- $\kappa > 0$ is called the *thermal diffusion coefficient* (since the Laplace operator models a (heat) diffusion process)

Boundary Conditions

Dirichlet boundary conditions:

- fix T on (part of) the boundary

$$T(x, y, z) = \varphi(x, y, z)$$

Neumann boundary conditions:

- fix T 's normal derivative on (part of) the boundary:

$$\frac{\partial T}{\partial n}(x, y, z) = \varphi(x, y, z)$$

- special case: insulation

$$\frac{\partial T}{\partial n}(x, y, z) = 0$$

Part III: Discretization: Finite Difference and Finite Volume Methods

From Continuous Back To Discrete Models

Finite Difference Methods

Meshes for Finite Difference Discretisation
Finite Difference Discretisation
Resulting Linear System of Equations
Discretisation Stencils

Finite Volume Methods

Finite Volume Meshes
Finite Volume Discretisation

From Continuous Back To Discrete Models

Continuous Models:

- result from a limit process ($h \rightarrow 0$) from discrete model (wire mesh, finite volume)
- opposite route \rightarrow discretisation

Discretisation methods:

- Finite Differences:**
“replace derivative by difference quotients”
- Finite Volumes:**
compute fluxes on boundary of control volumes and examine conservation laws

The Model Problem

- 2D Poisson Equation:

$$\frac{\partial^2}{\partial x^2} u(x, y) + \frac{\partial^2}{\partial y^2} u(x, y) = f(x, y)$$

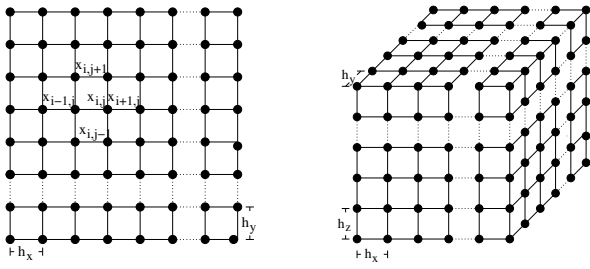
on the unit square $\Omega = (0, 1)^2$

- with Dirichlet boundary conditions:

$$u(x, y) = g(x, y) \quad \text{on } \partial\Omega$$

Meshes for Finite Difference Discretisation

- regular, Cartesian mesh; analogous to the wire-mesh model:



- compute approximate value of u for each mesh point:

$$u_{ij} \approx u(x_{ij}) \quad u_{ijk} \approx u(x_{ijk})$$

Finite Difference Discretisation

- replace partial derivative (at each mesh point) by difference quotient:

$$\frac{\partial^2 u}{\partial x^2}(x_{i,j}) \approx \frac{u(x_{i+1,j}) - 2u(x_{i,j}) + u(x_{i-1,j}))}{h_x^2}$$

$$\frac{\partial^2 u}{\partial y^2}(x_{i,j}) \approx \frac{u(x_{i,j+1}) - 2u(x_{i,j}) + u(x_{i,j-1}))}{h_y^2}$$

- leads to Linear System of Equations ($h := h_x = h_y$):

$$\frac{1}{h^2} (u_{i+1,j} + u_{i,j+1} - 4u_{i,j} + u_{i,j-1} + u_{i-1,j}) = f(x_{i,j}) \quad x_{i,j} \in (0,1)^2$$

$$u(x_{i,j}) = g(x_{i,j}) \quad x_{i,j} \in \partial\Omega$$

Resulting Linear System of Equations

- matrix-vector notation of the system:

$$A_h x_h = f_h$$

- x_h a vector of all unknowns u_{ij}
⇒ requires *numbering* of the unknowns
- using row-wise numbering, e.g.:
 $x_h = (u_{1,1}, \dots, u_{1,n}, u_{2,1}, \dots, u_{2,n}, \dots, u_{n,1}, \dots, u_{n,n})$

Resulting Linear System of Equations (2)

- A_h is a sparse matrix (only 5 diagonals are non-zero)
- A_h is block-tridiagonal:

$$A_h = \frac{1}{h^2} \begin{pmatrix} B_h & I & & & \\ & I & \ddots & & \\ & & \ddots & \ddots & \\ & & & \ddots & I \\ & & & & I & B_h \end{pmatrix}$$

- $B_h = \text{tridiag}(1, -4, 1)$, where I is the unit matrix

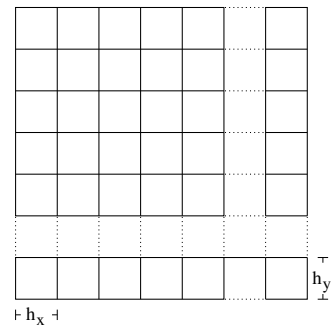
Notation: Discretisation Stencils

- idea: illustrate the matrix structure via a so-called **discretisation stencil**
- represents one row of the matrix
- matrix elements ordered according to their "geometrical" orientation
- discretisation stencil for Poisson equations:

$$1D: \frac{1}{h^2} [1 \quad -2 \quad 1] \quad 2D: \frac{1}{h^2} \begin{bmatrix} 1 & & \\ & -4 & \\ & & 1 \end{bmatrix}$$

Finite Volume Methods – Meshes

- domain Ω subdivided into grid cells/elements Ω_{ij} :



- u constant in Ω_{ij} , i.e., $u(x, y) = u_{ij}$

Finite Volume Discretisation

- integrate over grid cells Ω_{ij} :

$$\int_{\Omega_{ij}} \frac{\partial^2 u}{\partial x^2}(x, y) + \frac{\partial^2 u}{\partial y^2}(x, y) dx dy = \underbrace{\int_{\Omega_{ij}} f(x, y) dx dy}_{:= f_{ij} h_x h_y}$$

- integration by parts:

$$\int_{\Omega_{ij}} \frac{\partial^2 u}{\partial x^2}(x, y) dx dy = \int_{y_{j-\frac{1}{2}}}^{y_{j+\frac{1}{2}}} \left(\frac{\partial u}{\partial x}(x, y) \Big|_{x_{i-\frac{1}{2}}}^{x_{i+\frac{1}{2}}} \right) dy$$

$$\int_{\Omega_{ij}} \frac{\partial^2 u}{\partial y^2}(x, y) dx dy = \int_{x_{i-\frac{1}{2}}}^{x_{i+\frac{1}{2}}} \left(\frac{\partial u}{\partial y}(x, y) \Big|_{y_{j-\frac{1}{2}}}^{y_{j+\frac{1}{2}}} \right) dx$$

Finite Volume Discretisation (2)

- remember: u constant in Ω_{ij} , i.e., $u(x, y) = u_{ij}$
- thus approximation of derivatives on edges:

$$\frac{\partial u}{\partial x} \Big|_{x_{i+\frac{1}{2}}} = \frac{u_{i+1,j} - u_{ij}}{h_x} \quad \frac{\partial u}{\partial x} \Big|_{x_{i-\frac{1}{2}}} = \frac{u_{ij} - u_{i-1,j}}{h_x}$$

$$\frac{\partial u}{\partial y} \Big|_{y_{j+\frac{1}{2}}} = \frac{u_{i,j+1} - u_{ij}}{h_y} \quad \frac{\partial u}{\partial y} \Big|_{y_{j-\frac{1}{2}}} = \frac{u_{ij} - u_{i,j-1}}{h_y}$$

- again leads to Linear System of Equations:

$$\frac{1}{h_x} (u_{i+1,j} - 2u_{ij} + u_{i-1,j}) h_y$$

$$+ \frac{1}{h_y} (u_{i,j+1} - 2u_{ij} + u_{i,j-1}) h_x = f_{ij} h_x h_y$$

Finite Volume Discretisation – More General ...

- typical formulation for first-order PDEs:

$$\int_{\Omega_j} \frac{\partial u}{\partial t} + \frac{\partial F(u(x, y))}{\partial x} + \frac{\partial G(u(x, y))}{\partial y} dx dy = \dots$$

- and analogously:

$$\int_{\Omega_j} \frac{\partial F(u(x, y))}{\partial x} dx dy = \int_{y_{j-\frac{1}{2}}}^{y_{j+\frac{1}{2}}} F(u(x, y)) \Big|_{x_{j-\frac{1}{2}}}^{x_{j+\frac{1}{2}}} dy$$

$$\int_{\Omega_j} \frac{\partial G(u(x, y))}{\partial y} dx dy = \int_{x_{j-\frac{1}{2}}}^{x_{j+\frac{1}{2}}} G(u(x, y)) \Big|_{y_{j-\frac{1}{2}}}^{y_{j+\frac{1}{2}}} dx$$

- for Poisson Equation: $F(u) = \frac{\partial}{\partial x} u$, etc.